

N 63 19595

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VOLUME 1 — FINAL REPORT
for Period February 26, 1962 to February 25, 1963

STUDY OF DYNAMIC AND STATIC SEALS
FOR LIQUID ROCKET ENGINES

NASA Contract No. NAS 7-102

OTS PRICE

XEROX \$ ____

MICROFILM \$ 1,469,5

GENERAL 6 ELECTRIC

TABLE OF CONTENTS

		Page
I. Summary		1
II. Descrip	otion of Study Program - R. C. Elwell	4
III. Analys	sis of Results of Program - R. C. Elwell	8
-Appendix-	Evaluation of Currently Available Sealing Methods-A.J. Bialous	13
	Recommended General References	17
	List of Tables	18
	Tables 1 through 21	

Final Report Distribution List

DESCRIPTION OF PROGRAM

AND

RESULTS OF EVALUATION OF CURRENTLY AVAILABLE SEALING METHODS

R. C. Elwell A. J. Bialous

Volume 1 - Final Report

March 29, 1963

Study of Dynamic and Static Seals for Liquid Rocket Engines

Contract Period - February 26, 1962 to February 25, 1963

TO: Chief, Liquid Propulsion Systems, RPL

National Aeronautics and Space Administration

400 Maryland Avenue, S.W.

Washington 25, D.C.

PREPARED BY: Advanced Technology Laboratories

General Electric Company Schenectady, New York

CONTRACT NO: NAS 7-102

CONTRACTING DEPARTMENT: Missile and Space Division

Philadelphia, Pennsylvania

I. SUMMARY

19595

This is the first volume of a report which describes the technical efforts applied in the performance of Contract NAS 7-102, between the National Aeronautics and Space Administration and the General Electric Company (Missile and Space Division). The technical work was accomplished by personnel of the Advanced Technology Laboratories, (formerly the General Engineering Laboratory), of the General Electric Company.

The title of this program is "Study of Dynamic and Static Seals for Liquid Rocket Engines." The objective of the program is to advance the technology of dynamic and static seals for liquid rocket engines of all types.

In the period being reported on here, the main elements of the technology of sealing have been defined, and the present state of knowledge in most of these areas established.

Studies have been conducted in several of these areas, as described below. In addition, extensive surveys have been conducted of two main bodies of technical literature—-U.S. Government reports and open literature. The data derived from this task is being analyzed to establish the detailed capabilities of existing sealing methods. It has been found necessary to perform these two efforts because of the disorganized state of the literature in this technology.

The results of these various studies are presented in this report, which consists of four separate documents, summaries of which follow.

<u>Volume 1</u> (This document) - <u>Description of Program and Evaluation of Currently</u>

<u>Available Sealing Methods</u>

The first part of this publication gives an overall description of the study, including its objective, scope, strategy and tactics. An analysis of the results of the study in overall terms is also included.

An appendix contains tables of data thus far reduced from published information. These tables are intended to furnish a quick guide to the capabilities of any particular sealing device.

Volume 2 - Studies on Special Topics in Sealing

Detailed reports are presented here on the specific individual studies carried out on the following subjects:

- A. Gas Permeation Through Solids
- B. General Seal Analysis
- C. The Labyrinth Seal
- D. Leak Detection and Measurement
- E. Cryogenic Sealing
- F. Two Phase Phenomena in Dynamic Face Seals
- G. Seal Materials

Volume 3A - Bibliography of ASTIA Literature on Seals

This publication contains 682 abstracts of government reports available from the Armed Services Technical Information Agency (ASTIA). A complete Subject Index in the back allows rapid searching for references on particular topics. Definitions are included.

Volume 3B - Bibliography of Open Literature on Seals

This publication contains 1482 abstracts of references available to the general public on the subject of sealing. As in the case of the ASTIA bibliography in Volume 3A, a complete subject index with definitions is included. With Volume 3A, this constitutes the most extensive bibliography on this subject known to exist, and is the final document in this report.

Project Engineer is R. C. Elwell. Project Manager is G. R. Fox. This study is being conducted within the Mechanical Engineering Laboratory, which is one of the Advanced Technology Laboratories.

NASA Technical Manager is R. S. Weiner, Liquid Propulsion Section, Jet Propulsion Laboratory. NASA Project Manager is F. E. Compitello, Code RPL, Office of Liquid Rockets.

II. DESCRIPTION OF PROGRAM

A. Objective

The objective of this program is to advance the technology of dynamic and static seals for liquid rocket engines. A study of the fundamentals of sealing processes is to be carried out, in order to provide the understanding necessary to formulate design criteria for seals in future liquid rocket engines.

B. Scope

The scope of this program may be represented diagrammatically as follows, with the shaded areas representing the areas of interest in this study:

Scope of Study of Dynamic and Static Seals for Liquid Rocket Engines

	STATIC SEALS	DYNAMIC SEALS
Launch Vehicles	Connectors	
Space Vehicles		

The study of "zero leakage" connectors for launch vehicle engines is the subject of a separate NASA study at ATL.

C. Strategy

Our basic strategy in attaining the above objective may be described in simplified fashion as follows:

 Reduce the sealing technology to its main technical areas, for further detailed study in each area. This essentially established the "road map" of the project. The reduction we are using for working purposes appears in Figure II-1 of this Section.

- 2. Acquire all information possible on static and dynamic sealing technology, and organize this so that the present state of the art may be apparent in all these areas. This required a comprehensive literature search, since this had never been done, and the results of prior work in this field were scattered in many places.
- 3. Appraise the state of existing knowledge and experience in each area, on the basis of the information acquired in the previous step. This may be represented pictorially as in Figure II-1, which presents our present estimates of the state of each area.

The following working definitions are being employed as standards in the study:

- a. Poor: Basic principle unknown. Empirical data serves as basis for design and use.
- b. Fair: Basic principle known. Analytical techniques for performance prediction not available.
- c. Good: Analytical prediction of performance possible with rough

 (say + 25%) accuracy.
- d. Excellent: Performance analytically predictable with high accuracy (say ± 5%).
- 4. Simultaneously with the previous step, establish how much knowledge will be needed in each area for the design of future liquid rocket engines. The overall length of the bars shown in Figure II-lrepresent

our current opinions.

5. Furnish the knowledge necessary to <u>advance</u> the technology in the areas where this has been determined to be necessary in the previous steps. Whereas the previous tasks may be regarded as defining the present position of the technology, they cannot advance it by themselves. Therefore, this final step is required in the overall strategy in order to attain the objective.

D. Tactics

In general, our program tactics may be described as carrying out a broad advance across the field illustrated in Figure II-1, and at the same time advancing the technology in specific areas where we know definitely that an advance is necessary and feasible to attain. Volume 2 presents the results of most of these studies. The Appendix of this volume contains the results of the evaluation of present sealing techniques as it now stands. Volume 3 presents the results of our study of the literature.

Figure II-1

MATRIX OF STUDY OF DYNAMIC AND STATIC SEALS FOR LIQUID ROCKET ENGINES

			POOR	FAIR	GOOD	EXC.
STATIC SEALING	А. В.	Selection and Control of Geom. 1. Positive Contact 2. Controlled Clearance Control of Fluid Properties 1. Freeze 2. Magnetic 3. Other Types Control of Fluid Forces				J
DYNAMIC SEALING	А. В.	Rotary Motion 1. Selec. and Cont. Geom. 2. Control Fluid Properties 3. Control Fluid Forces Oscillatory Motion 1. Selec. and Cont. Geom. 2. Control Fluid Properties 3. Control Fluid Forces Reciprocating Motion 1. Selec. and Cont. Geom. 2. Control Fluid Forces Reciprocating Motion 1. Selec. and Cont. Geom. 2. Control Fluid Properties 3. Control Fluid Forces]
GENERAL AREAS	A. B. C. D. E. G.	Leak Detection and Meas. Materials Problems Cryogenic Fluids Sealing Permeation Flow Phenomena Fundamental Analytical Work Literature Other	//////////////////////////////////////	77777777		

Estimate of Advance Required for Future Liquid Rocket Engines

III. ANALYSIS OF RESULTS OF PROGRAM

A. General

In our discussions with technical personnel in various aerospace industry and government establishments, propellant leakage was consistently described as a major problem. Also, our efforts to effect an improvement in fundamental knowledge of sealing were generally encouraged, because of the difficulty experienced in solving problems without it. It appears that the condition of the technology as we found it was largely due to a history of one-shot problem solving programs which contributed little knowledge to the field.

Our impression is that the problem of generating this fundamental knowledge has been too big to be accomplished in the small scale efforts usually applied to seal problems. Unless this situation is corrected, particularly in regard to analytical ability, the empirical solve-one-problem-at-a-time approach is going to be extremely costly in time and money in the future, when propulsion systems get larger and more complex.

Overall, we have found the technology of fluid sealing to be in a rather disorganized state and not as technically mature as many others such as heat transfer or lubrication, to cite familiar examples. There are no books, for example, except for two small volumes in German, and until the inception of this study, there was not even a bibliography on the subject.

One of the major results of this program, prosaid though it may sound, was the establishment of the main elements of the technology (See Fig. II-1), previous page) in an orderly structure. We do not claim that this outline is final or perhaps even the best organization, but it will at least serve the purpose of presenting a "target," or example, for future improvement, and serve as a framework for discussion.

D. Gas Permeation Through Solids (See Vol. 2)

In storing gases for long periods of time, such as in some projected spacecraft, losses due to this process could be significant. Our study of the state of the art is presented in Volume 2. It is concluded that more experimental data is required in order to adequately predict long time flow losses because of this.

E. General Seal Analysis (See Vol. 2)

It has been necessary to start on a very fundamental level in this area, because the sealing technology suffers from a particular weakness in analytical techniques. Derivation of the basic equations for analysis of flow between concentric vertical cylinders and parallel plates is presented, using tensor notation.

It is hoped that these basic derivations will serve as a foundation for future specialized analyses of many types of sealing devices, in a fashion analogous to the Navier-Stokes equations in fluid mechanics.

F. The Labyrinth Seal (see Vol. 2)

A survey was conducted of present knowledge and design ability on this device, which is particularly important because of its wide use in liquid rocket engine turbomachinery. It is found that design information on this type of seal is in a much better state than most other dynamic sealing devices.

G. Leak Detection and Leakage Measurement (See Vol. 2)

A rather extensive report was prepared on the state of the art in this area, which is directly related to sealing in that evaluation of seal effectiveness is just as important as design ability. Topics covered include types of leakage flows, conversion of flow rates to standard conditions, and leakage

A further accomplishment along these organizational lines was the preparation of an extensive group of definitions. These are placed for convenience in the Subject Indexes of Volumes 3A (pp. 187-208) and 3B (pp. 147-161). By incorporation of the subsequent suggestions and criticism expected after issuance of this report, this list should serve a valuable function in the potential establishment of standard terms.

B. Literature on Seals and Sealing (See Vol. 3)

For a technology which has been of such wide interest over the years, the state of its literature as we found it was a source of amazement. By coincidence, two excellent bibliographies of limited scope appeared shortly after we initiated an exhaustive survey (See Vol. 3B, pp. iii). These have been incorporated in Volume 3B, which goes back to the original publication of 23 major sources in the English language.

This effort alone will save countless man-hours of random literature searching in the future, not only by designers of liquid rocket engines, but by other industry and government personnel as well.

C. Evaluation of Currently Available Sealing Methods (See Vol. 1)

There is no book on this subject except as noted above, and one of those is 20 years old. A person setting out to design a seal or establish operating capabilities of a particular device, therefore, has no reliable place to look for what is available and what has been accomplished to date.

The Appendix of this volume is an interim report on the data we have been able to evaluate thus far in an attempt to produce a quick design reference. This effort could not be completed within this period because of the condition of the literature as discussed above. In this task, certain general references have been found useful and these are pointed out in this appendix.

testing methods. A considerable amount of work remains to be done on this topic, especially if sealing capabilities improve, which will require better standards and techniques.

H. Cryogenic Sealing (See Vol. 2)

This topic was singled out for a special state of the art evaluation due to the wide use of cryogenic propellants in both present and future engines. It was concluded that uniqueness of sealing problems in this area is mainly due to friction and wear of dynamic seals in atmospheres of propellant gases. A description of major current activities in this field is included.

I. Two Phase Phenomena in Dynamic Face Seals (See Vol. 2)

These devices are even more widely used than labyrinth seals (Appendix C) in rocket engines and are not understood to any significant extent. Most theoretical work has been concerned with the problem of <u>fluid film load</u> capacity in the device, to the apparent detriment of theories on the subject of the mechanism by which it <u>seals</u>. In this section a new theory of its function is presented, based on two phase fluid phenomena.

It is hoped that this will generate new ideas on this device, which frequently causes operational problems in dynamic sealing.

J. Seal Materials (See Vol. 2)

A limited effort was carried out in this area, and the results are documented in Appendix G of Volume 2. A limited number of materials have been used in most sealing devices, so that among the large number of other available materials there may be potentially superior ones untried thus far. A more extensive study is required in this area. A continuation of the evaluation of existing sealing techniques (see Appendix, this volume) is expected to document the most successful seal materials in more detail, and assist in this task.

There is of course considerable interrelation between several of the above studies. For instance, materials considerations are important to the studies on gas permeation, the labyrinth seal, cryogenic sealing, and the occurrence of the two phase phenomena in face seals. For another example, the occurrence of the two phase phenomena will ultimately be predictable through the results of the general analysis. Leakage measurement, of course, applies to all sealing applications.

APPENDIX

EVALUATION OF CURRENTLY AVAILABLE SEALING METHODS - A. J. BIALOUS

This is an interim report of an analysis and evaluation of published data on all types of sealing devices. The analysis has been conducted and the report prepared by the Advanced Technology Laboratories of the General Electric Company under NASA contract number NAS-7-102. The objective of this analysis is to establish the state of present knowledge of various sealing devices and to present this information in a clear, concise and convenient manner.

The primary source of information for this analysis has been the published results obtained in <u>testing</u> programs conducted on sealing devices. A few selected <u>recommendations</u> made by persons and organizations considered to be eminent in this field or in related subjects, have also been included. Data presented is limited to that obtained during tests in which the seal has been considered to perform in a <u>satisfactory</u> manner under stated application and environmental conditions.

The published information initially reviewed was in abstract form and is in general, limited to the abstracts listed in Volume 3A of this Final Report. These are unclassified abstracts which have been supplied by ASTIA (Armed Services Technical Information Agency) on the subject of static and dynamic seals. A few of the open-literature abstracts in Volume 3B of this Final Report have also been reviewed. In this latter area, considerable emphasis has been given to general seal references.

Each of the approximately 700 listed ASTIA abstracts was reviewed and analyzed for applicability. It had been estimated that about half of the

necessary data would be obtained from these abstracts, but the dearth of information given in a substantial number of them requires that the complete reference be read so that information of sufficient depth can be obtained.

Most of the ASTIA reports which have been selected for reading have been accumulated. Experience, based primarily on the review of ASTIA abstracts indicates the following findings as oriented to this program:

- 1. Abstracts which contain useful information----- 5%
- 2. Promising references to be read------17%
- 3. Abstracts containing no significant information------78%

 A review of the data documented to date shows that 50% of the entries were made directly from abstracts and that in the remaining 50% of the cases, the original references were studied. A disciplined effort has been made to present only the facts given in the literature and not to make any interpretations, extensions or generalizations of these facts.

The documented data is presented in a handbook type of format (Tables 1-21) which permits rapid performance data evaluation. Information obtained from ASTIA reports can be identified in the tables by the ASTIA number which is usually preceded by the letters AD. Open literature sources are identified by the serial numbers assigned to them in Volume 3B. These numbers are preceded by the first letter of the author's last name.

Seal types listed, and definitions used, conform with those given on pp. 147 to 156 of Volume 3B of this Final Report. Information concerning the applicability of a particular type of sealing device for a specific system application, can be found in the tables under the heading of the sealing device for either static or dynamic application. (See List of Tables preceding Table 1). For example, performance data for O-Ring sealing methods used on rotating shafts are listed under O-Ring, Dynamic Seals in Tables 14

to 16.

Plans have been made to continue this analysis in greater depth.

Included in these plans is the review of the approximately 75 selected ASTIA papers and the examination of the 1500 open-literature abstracts listed in Volume 3B of this Final Report.

Whereas the information presented in Tables 1-21 is organized on a seal-oriented basis for working convenience, ultimately it will be organized on a user-oriented basis. This user-oriented approach will permit a search to be concentrated on the sealing category of interest and will ignore other capabilities of a particular sealing device.

SEAL CATEGORIZATION SYSTEM

I. DYNAMIC SEALS

C. Reciprocating Motion 1. Selection and Control of Geometry a. Positive Contact i. Packings ii. Lip iii. Rings iv. Diaphragm b. Controlled Clearance i. Labyrinth	<pre>ii. Bushing 2. Control of Fluid Properties a. Freeze b. Magnetic 3. Control of Forces</pre>	B. Control of Fluid Properties 1. Freeze 2. Magnetic C. Control of Forces
, O @ +	ii. Bushing 2. Control of Fluid Properties a. Freeze b. Magnetic 3. Control of Forces	A. Selection and Control of Geometry (Continued) iii. Hermetic Sealing Welding Brazing Soldering Cementing Mold in place seals 2. Controlled Clearance ii. Labyrinth iii. Bushing
# t	2. Control of Fluid Properties a. Freeze b. Magnetic a. Control of Forces a. Centrifugal b. Screw c. Magnetic	A. Selection and Control of Geometry 1. Positive Contact i. Elastic Deformation Gaskets Rings Diaphragms ii. Plastic Deformation Gaskets Rings

RECOMMENDED GENERAL REFERENCES

The Seals Book - Jan. 19, 1961, Produced by Machine Design, The Penton Publishing Co. - Penton Building, Cleveland 13, Ohio.

SAE Aerospace Standards, Recommended Practices, Information Reports Cooperative Engineering Program - Society of Automotive Engineers, 485
Lexington Avenue, New York 17, N. Y.

Surface Texture - ASA-B46.1-1962, Surface roughness, waviness and lay.

Surface Roughness, Waviness and Lay-Military Standard - MIL-STD-10A.

<u>Surface Roughness</u> - SAE Aeronautical Standard AS291B - Surface Roughness

<u>Surface Finish Control</u> - SAE Recommended Practice SAEJ449.

The Accuracy of Surface Roughness Assessment by P. F. Jones - Microtechnic Vol. XVI No. 3, pp. 105-112.

Scientific Foundations of Vacuum Techniques by Saul Dushman, Second Edition 1962, John Wiley Publishing Co.

LIST OF TABLES

SEAL TYPE	TABLE NO.
Bushings	1
Caulking (Static)	2
Face or Axial	3
Gasket (Static Seal)	4
" (" ")	5
" (" ")	6
Hermetic	7
Lip or Radial (Dynamic)	8
Packing-Stuffing (Dynamic)	9
Packing-Stuffing Box (Static)	10
" - " " (")	11
Misc. Rings (Dynamic)	12
O-Ring (General)	13
O-Ring (Dynamic Seal)	14
" (")	15
" (")	16
O-Ring (Static)	17
" ("")	18
" (")	19
"V"-Ring (Dynamic)	20
Special Sealing Concepts (Static)	21

Bushings				Gradient 28 to 1355	1	12000 to 16000 RPM	•	•	Δ P = 300					To Decomposed	reroxide Gas 18 Min.		Decomposed Peroxide Gas Deteriorated	Glass-Teflon Seal		
				Gradient 18 to 75		14000 to 17800 12000 to 16000 RPM	Teflon-Glass	Electrolized	$\Delta P = 250$				30	To Air	30 Min.			000		
SEAL TYPE				Gradient 25 to 1310		10000 to 16000 RPM			$\Delta P = 250$					To Decomposed	reroxide Gas 30 Min.	Turbo-Pump ooled Bearing	<u></u>			
	H.O. Pederson	AD225213	In Fixture	Gradient 34 to 75	Rotation	16400 to 17300 RPM	'S'' Seon		ΔP = 125				107	90% Hydrogen Peroxide to Air To Decomposed	5 Min.	Rocket Engine Turbo-Pump System Fluid Cooled Bearing				
NALYSIS	B. L. Johnson	AD201739	X		Pump - 4"	4 Stage 1700 to 2100 RPM	Carbon Sleeve Stellite Face	with Synthetic Rubber O-Rings Steel	600 Rated				No Leakage	aiter Run-in Military Gas- oline	1000 Hrs.	More than 1000 hrs. Sleeve Type Mechanical Seal. Mfg.	by J. Crane Co.		COIL SPRING	
SEAL ANALYSIS	REFERENCE	NO.	TYPE OF INFORMATION TESTS RECOMMENDATION	TEMPERATURE ^O F MAX. RANGE	MOTION-TYPE	VELOCITY FPM	MATERIAL SEAL	MATE	PRESSURE PSIA MAX.	RANGE	LOADING SPRING COMPRESSION	SURFACE FIN. (µ in CLA)	LEAK RATE(Atm. cc/sec)	FLUID	TEST DURATION	LIFE REMARKS	DESIGN			

SEAL TYPE Caulking (Static)

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<	Ç
NO.	4
U	2

Bashfield, W. H. Dec. 1955 AD94590	×	140			Glass	Solder 97 1/2% P _b	2 1/2% Ag 0.7	, <u> </u>				Air	Satisfactory at	500 V. RMS		
R. W. Bryant et al July 1958 AD200898	×	cycling			Threaded Joint RD1286 Cement RD1284 Luting	MK8	4 1/2 Ft Drop to concrete				-				Avoid Dissim- ilar metals	 Use Shoulder on Threaded
Pigman, S. S. April 1958 AD200230	×	500			Silicone Base Sealer	Insulated Cable						Air		96 hrs. Little Change	ın Insul, Resis	_
Scharfenstein, C.F. May 1958 AD218038	×				Min. Mining Chrome-Lock Tape Type N-	MÍL-G 20241, Symbol 2291 PRC 1201T	to Al; Fe to Al Joint					10% Aqueous	30 days plus	200 hrs. Accel.	Weathering Most Satisfac- tory Method	
REFERENCE NO.	TYPE OF INFORMATION TESTS RECOMMENDATION	TEMPERATURE OF MAX. RANGE	MOTION-TYPE	VELOCITY FPM	MATERIAL	MATE	PRESSURE PSIA	RANGE	LOADING SPRING COMPRESSION	SURFACE FIN. (in. CLA)	LEAK RATE (Atms. cc/sec)	FLUID	TEST DURATION	LIFE	REMARKS	DESIGN

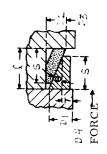
on Threaded Part See Also ADI54972

Curren, A.N. H.O. Pederson et al Jan. 1957 et al Feb. 1959 — AD19437 AD225213	RMATION X In Fixture	2 OF 47 to 91 40 to 140	Rot Rotation Approx.	M 10,000 16,000 rpm.	Carbon-Base Aluminum Rings Oxide Chromium 25% Glass Plate Metal Filled Teflon	A 38.3	25 to 160 15 to 25	NO	(in, CLA)	Zero Inlet Seal	Air 90% H ₂ O ₂	Wear 0005"/ hr. (Ring) COOLING AIR TO	TURBOJET BLADES PRESSUR	CALLED CALLED		SUPPLY FO	SUPPLY FOR
REFERENCE NO.	TYPE OF INFORMATION TESTS RECOMMENDATION	TEMPERATURE OF MAX. RANGE	MOTION-TYPE	VELOCITY FPM	MATERIAL SEAL MATE	PRESSURE PSIA MAX.	RANGE	LOADING SPRING COMPRESSION	SURFACE FIN. (in. CLA)	LEAK RATE (Atms. cc/sec)	FLUID	TEST DURATION LIFE REMARKS	DESIGN ROTATING PLATE		GRAPHITE - BASE	GRAPHITE- BASE RINGS STATIONARY	GRAPHITE- BASE RINGS STATIONARY

1/16" GASKET

SEAL ANALYSIS	ALYSIS		SEAL TYPE		Gaskets (Static Seal)	1)		TABLE	ιc
REFERENCE NO.	M. B. Donald 1955 D057	R. Brymmer June 1959 B124			Met. Vickers Sept. 1949 MV9804			J. Holden, et al.June 1959H060	J. W. Hull Sept. 1960 H079
TYPE OF INFORMATION TESTS RECOMMENDATION	×	×	×	×	×	×	×	×	×
TEMPERATURE ^O F MAX. RANGE MOTION-TYPE	70F	0 to 400C (32 to 752)	-188C-800C .		194			20 to 550C (68 to 1022)	-65 to 700
VELOCITY FPM									
MATERIAL SEAL	Rubber Bonded Asbestos Fiber	s.s.	Soft Iron	Cu, Fe	Lagnite	Neo-K-Tex	Duram	A1	Tool & SS
MATE	Metal Metal	Mild Steel			Metal		-	Steel	Nitraloy, Tool
MAX.	1615				29.7	14.7	24.7		St. 4000
RANGE		0 to 14.7						0 to 14.7	Surge 6000
LOADING SPRING COMPRESSION								To 4100 lb/in	
SURFACE FIN. (µ in.CLA)								Fine Emery	8 to 32
LEAK RATE (Atm. cc/sec)	0	10-6	10-9	10-8	0	0	0	on King 0	0
FLUID	Water	Air		A	Oil			Air	Hydraulic Fluids
TEST DURATION					10 days	7 days	20 Hrs.	10 Cycles	-
REMARKS									Design Formu- lae and Tables are given
DESIGN	_								0

 $\theta = 15^{\circ} \text{ to } 20^{\circ}$ $D_{4} = D_{1}-2 \text{ (S TAN } \theta + \text{T)}$ $D_{3} = D_{2}-2 \text{ (} \text{£ TAN } \theta + \text{T)}$ SEE (H079)



9

Gaskets (Static Seal)	(magazina) mariana
SEAL TYPE	
NALYSIS	
SEAL ANAI	

Gaskets (Static Seal)		00 etat, reb. 1951 0 AD204937	×	1d 530 to 1025			_	Flat Flange	ng 900 Held	0	40, 000 in/lbs Bending Moment		ling			Can Tolerate 50 to 100 Cycles	of Heat Shock	
SEAL TYPE	M. Sabanas	AD238020	×	1000 Held			Molybdenum Fibers Impreg- nated with Ag		Pulsating	0 to 5000		,	Good Sealing	Air				
SEAL	4		×						3000		Precondition 36 Days,220F in Air	ı) Air	ı			
			×				Hycar 1041 Plus Hycar 1312 (Acrilonitriles) In Pipe & Tube Union		2000		econditioned at 220 F in Test Fluid for 36 Days			Petroleum Base (MIL-0-5606) or Phosphate Ester	nunne) 60 Days			
SEAL ANALYSIS	A. E. Barrett	AD145361	*				Hycar 1041 P (Acrilo In Pipe &		10, 000		<u> </u>	7		Petroleum Ba	1/2 Hr.			
SEAL A	REFERENCE	NO.	TYPE OF INFORMATION TESTS RECOMMENDATION	TEMPERATURE OF MAN. RANGE	MOTION-TYPE	VELOCITY FPM	MATERIAL SEAL	MATE	PRESSURE PSIA MAX.	RANGE	LOADING SPRING COMPRESSION STIPLAGE FIN (1, 1, CLA)	TEAK BATE (Atm 00/00)	THE CAME OF SEC.	FLUID	TEST DURATION	LIFE	REMARKS	

Society of Automotive Engineers: ARP 748, ARP 453;
Aerospace Standards - Cooperative Engineering Program (See General References)
American Standards Association: ASA-B16.21 -1962 Nometallic Gaskets for Pipe Flanges
ASA-B16.20-1956 Ring-Joint Gaskets and Grooves For Steel Pipe Flanges
Chemical Engineering Oct. 1, 1962, pp. 83-88, "Select and Apply Gaskets Effectively" by J.J. Wahlen

Bashfield, W. H. Dec. 1955 AD94590	×	140			glass	Solder 97 1/2% Pb 2 1/2% Ag	0.7					Air	Satisfactory at 500V rms	
REFERENCE NO.	TYPE OF INFORMATION TESTS RECOMMENDATION	TEMPERATURE OF MAX. RANGE	MOTION-TYPE	VELOCITY FPM	MATERIAL SEAL	MATE	PRESSURE PSIA MAX.	RANGE	LOADING SPRING COMPRESSION	SURFACE FIN. (in, CLA)	LEAK RATE (Atms, cc/sec)	FLUID	TEST DURATION LIFE REMARKS	

DESIGN

SEAL TYPE Lip or Radial (Dynamic)

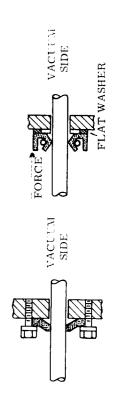
 ∞

	×	30	Reciprocating (2")	†	Buffed Steel	↑				1.5×10^{-5}		s. per stroke	B" Oil
wton	×	120C 248		8								200 hrs.	Apiezon "B" Oil
R. H. V. M. Dawton Oct. 1957 D-014	×	120C 248	Rotation	6 1 Rev/sec, 3/8 diam. Neoprene	Steel Glass	Brass	0-14.7			5×10^{-6}	Air	2 days	Lubrication;
REFERENCE NO.	TYPE OF INFORMATION TESTS RECOMMENDATION	TEMPERATURE OF MAX. RANGE	MOTION-TYPE	VELOCITY FPM MATERIAL SEAL	MATE	PRESSURE PSIA	RANGE	LOADING SPRING COMPRESSION	SURFACE FIN. (in. CLA)	LEAK RATE (Atms, cc/sec)	FLUID	TEST DURATION LIFE REVIARKS	

S.S.

20

×



DESIGN

per stroke

 4.5×10^{-5}

REFERENCES. SAE-JIIO "Bench Leakage Test for Automotive Radial Oil Seals For Rotating Shafts," SAE-JIII "Radial Seal Nomenclature; Paper No. 473A. Society of Automotive Engineers - "Engineering Facts About Lip Seals" by J. D. Symons, Jan. 1962 (S104)

6				7									
TABLE		×	400	Reciprocating-6" Travel at 30 Cycles per Min.	1X4439 Cumpac anville - 95% aconel Wire	1				2.6cc/1000 cyc		25, 000 Cycles 14 Hrs.	
R.B. Resek	July 1956 AD97258	*	80	Reciprocatir at 30 Cycle	Ring Packing (MX4439 Cumpac Seal) - Johns-Manville - 95% Asbestos with Inconel Wire Mesh Core	Cycling	700 10 2000	50 Lb-Ft		3.4 cc/1000 cyc. 2.6cc/1000 cyc.		5000 Cycles 3 Hrs.	
iamic)		×	550		Ring Packing (MX4439 Cumps 453-26C,1203-70Seal) - Johns-Manville - 95% Asbestos with Inconel Wire Mesh Core					•		1.5 Hrs.	
Packing-Stuffing (Dynamic)		×	200		453-9A							5 Hrs.	
		×	-65 to +400		453-26C					Satisfactory -	Hydraulic Oil		
SEAL TYPE		×	80	Test for Effect of 50,000 cy.at 80F	AN6227 - O-Ring Comp					38cc Total	Hydraulic Oil MIL-0-5606	50, 000 Cycles	
		×	-40 ire to 300F	mine Effect des at 300F		3000				.007		1 Hr.	
ANALYSIS F.R. Straus Jan. 1955	AD74089	×	-40 Prior Exposure to 300F	Tests to Determine Effect of 40, 000 Cycles at 300F	O-Ring Neo- prene WRT Compound with— Spiral Teflon	25 miles				0	Hydraulic Oil ML08200	20 Hrs.	
SEAL	NO.	TYPE OF INFORMATION TESTS RECOMMENDATION	TEMPERATURE ^O F MAN. RANGE	MOTION-TYPE VELOCITY FPM	MATERIAL SEAL MATE	PRESSURE PSIA MAX.	RANGE	LOADING SPRING COMPRESSION	SURFACE FIN. (µ in.CLA)	LEAK RATE (Atm. cc/sec)	FLUID	TEST DURATION LIFE REMARKS	DESIGN

S
YSI

SEAL TYPE Packing-Stuffing Box (Static)

mic)			×	600 -40 to 450			Hexafluoropro- pylene & Vinyl- idene Fluoride	Copolymers						Great Many Fluids &				
sc. Rings (Dynamic)			×	-100 to 350		. Rot. 350-600	Fluorosilicone Polymers							Pet.Based Fuel & Lub. Ester	Type Lub-Hy- drocarbons (Many)			-King
SEAL TYPE Misc.			×	-130 to 500		Not Gen. Recom.	tor Dyn. Uses Silicone Polymers							High Analine Oils-Hi Temp	Air-Ozone		DIFF TA BING	חברוםת
SEAL			×	-65 to 225	Oscillating		Polysulfide Polymer							Ozone, Pet. Base Fuels & Oils	Aromane solv.			
			×	-20 to 250	- Rotating - Osc		Butyl Rubber							Hyd.Fl.of Phos-Ozone, Pet. Base High Analine phate-Ester Fuels & Oils Oils-Hi Temp	Gases, Ozone Veg. Oils		X-RING	
ANALYSIS	M. H. Everett Jan. 1961	E034	×	-20 to 350F	Reciprocating -	Rot. 350-600	Polyacrylate- acrylic Acid Ester Polymer	Metal			$\stackrel{Peripheral}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}}{\overset{\epsilon_{\mathcal{H}}}}}}}}}}}}}}}}}$	370 16 to 32		Oil, Ozone			× ×	•
SEAL	REFERENCE	NO.	TYPE OF INFORMATION TESTS RECOMMENDATION	TEMPERATURE OF MAN. RANGE	MOTION-TYPE	VELOCITY FPM	MATERIAL SEAL	MATE	PRESSURE PSIA MAX.	RANGE	LOADING SPRING COMPRESSION	SURFACE FIN. (µ in CLA)	LEAK RATE(Atm. cc/sec)	FLUID	TEST DURATION LIFE REMARKS	DESIGN		

O-Ring (General)

SEAL TYPE

SEAL ANALYSIS

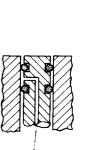
			SEAL	IIFE O-	-ning (General)	
REFERENCE NO.	R. D. Ford Sept. 1957 AD145846		C. E. Hamlin Mar. 1954 AD29114	A. MacCullen May 1960 AD241589		TABL
TYPE OF INFORMATION TESTS RECOMMENDATION	×	×	×	×		
TEMPERATURE ^O F MAX, RANGE	300	212	400 550-Short Time	275		
MOTION-TYPE VELOCITY FPM			Reciprocating 2 cps	Hydraulic Valve & Pump		
MATERIAL SEAL MATE	Silicone W96	Silicone W96	Neoprene W Al-Bronze Back-up	Buna N		
PRESSURE PSIA MAX.			3000	3000		
RANGE						
LOADING SPRING COMPRESSION						
SURFACE FIN. (μ in.CLA)						
LEAK RATE(Atm.cc/sec)	Satisfactory S	eal	Satisfactory	Not Abnormal		
FLUID	Air and ASTM Oil #2	Astmon #3	Orthosilicate Ester Hy-	Ordnite #8515 Hydraulic		
TEST DURATION LIFE REMARKS			draulic Fluid 100, 000 Cycles	Fluid 260 Hrs. Performance Satisfactory		
DESIGN	GENERAL REFERENCES: "Design Data for O-Rir Elastic Seals," by R.; Titon, W. R. Walker, Reports AD230658, Al AD265443.	NERAL REFERENCES: "Design Data for O-Rings and Similar Elastic Seals," by R. S. Roper, F. W. Titon, W. R. Walker, et al. in ASTIA Reports AD230658, AD230639, AD265443.	nd Similar oper, F.W. 1. in ASTIA 0639,	Oil and Gand Gand Gand Gand Gand Gand Gand	Oil and Gas Equipment - Dec. 1959 "O-Rings: Their Uses In Solving Important Sealing Problems" by J. R. Jordan Oil and Gas Equipment - Jan. 1960 "Choosing The Right O-Ring Can Solve Your Sealing Problem" by J. R. Jordan	Society Automoti Recommended Pr "Rubber O-Ring Seal and Packing

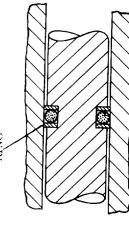
Society Automotive Engineers Recommended Practice - SAE J120 -"Rubber O-Rings for Automotive Seal and Packing Applications"

TABLE 14	C. E. Hamlin J. O. Bruno Mar. 1954 Dec. 1957 AD29114 AD118213	× ×	400 550 Short Time Above 160	Reciprocating 4" Recip.	2 cps 30 cpm	Neoprene "O"-Ring & "W" Teflon Liner AlBronze Dooling Control of the Property of the Propert			81	18 rms	Satisfactory 5 x 10-4 (10c/1000 cv.)	Orthosilicate MIL-0-5606 Ester Hyd. Oil	100, 000 cycles	O-RING CAP
	L. E. VanHise, etal,, Mar. 1959 AD68495	×		Rotating	2000 rpm	1/8" Buna N Hard Chrome- Plated Shaft	200		Spr. Retain. Ring	Microfinish				_
11)	R. L. Hayman L. E. VanHise, Jan. 1945 et al., Mar. 1955 H023 AD68495	×		Reciprocating		Synthetic Rubber Leather	3000							_
"O"-Ring (Dynamic Seal)	R. L. Hayman March 1945 H024	×		Swing Joint			200000 psi/sec Pulsating	0-3000						_
SEAL TYPE "O"-Rin		×	-40 to 600F -100 to 350F	4		Hexafluoropro-Fluorosilicate pylene & Vinyl- Polymers dene Fluoride	dness						rype Luby.	
	tt, et al. 1961	×	3 ⁰ F -130 to 500F			e Silicone Polymer	Copor 5000 PSI Varies with Ring Hardness			roove)		High Analine Oils, Pet. Prod	ucis, Og, Alf	COMMON DESIGN
SEAL ANALYSIS	M. H. Everett, et al. Jan. 19, 1961 E034	×	-65 ⁰ to 225 ⁰ F	Rotating	Max. 600	Polysulfide Polymer	5000 PSI Va			$\left \frac{1}{1} \right $ Max. 32 (G	(1)	Ozone, O ₂		_
SEAL A	REFERENCE NO.	TYPE OF INFORMATION TESTS RECOMMENDATION	TEMPERATURE OF MAX. RANGE	MOTION-TYPE	VELOCITY FPM	MATERIAL SEAL MATE	PRESSURE PSIA MAX.	RANGE	LOADING SPRING CONIPRESSION	SURFACE FIN. (µ in CLA) Max. 32 (Groove)	LEAK RATE (Atm. cc/sec)	FLUID	TEST DURATION LIFE REMARKS	DESIGN

"O"-Ring (Dvnamic Seal) SEAL TYPE SEAL ANALYSIS

SEAL A	SEAL ANALYSIS		SEAL	SEAL TYPE "O"-Rin	"O"-Ring (Dynamic Seal)	(al)				
REFERENCE	A.E. Barrett	R.D. Ford	A. E. Barrett	F. R. Straus				TABLE		_
NO.	AD252996	AD272431	July 1961 AD260465	Jan. 1955 AD74089					A	
TYPE OF INFORMATION TESTS RECOMMENDATION	×	×	×	×	×	×	×	×	×	
TEMPERATURE OF MAN. RANGE	180		601	-40	-40		500	550	80	
MOTION-TYPE	Reciprocating	Recip	160 Reciprocating			-65 to 400				
VELOCITY FPM		100 cpm								
MATERIAL	Viton B 70 & 90 Shore A	Viton B	Viton B 70 Shore A	Neoprene WRT O-Ring Spiral Teflon Back-up Rings on both	-Ring Spiral ings on both	453-26C O-Ring Comp. –	453-9A	453-9A 1203-70	AN6227	
MATE	Teflon Back-up		Teflon Back-up	sides)			Composition	
PRESSURE PSIA MAX.	3000		3000	25	3000					
RANGE		Cycle 0-3000								
LOADING SPRING COMPRESSION			•							
SURFACE FIN. (μ in CLA)										
LEAK RATE(Atm. cc/sec)	Satisfactory		Satisfactory	0	0.007	Satisfactory			38 cc	
FLUID	Phosphate Ester Base	Cellulube 220	Pet. Base Fluids	ML08200 Hydraulic Fluid	ic Fluid —				MIL-0-5606	
TEST DURATION LIFE REMARKS	r iuids	>5,000,000 cyc.		20 Hrs.	1 Hr.		5 Hrs.	1.5 Hrs.	50, 000 Cycles	
DESIGN						BACK-UP RING				
						/			-	





	1	- :		· · · · · · · · · · · · · · · · · · ·									1	-		_	
			×	-300 to 500			Teflon							Various Chemicals			
	TABLE 17		×	-20 to 325			Acrylate							Pet. Oils & Hydraulic Fluide	9		
			×	500 -130 to 400			Silicone							Air, Inert Gases- High Analine			
			×	-65 to 160			Buna S							Water Alcohols Kotones		1	~~~
O-Ring (Static)			×	-65 to 160			Butyl							Phosphate Ester			
			×	-65 to 170			Thiokol							Alcohols, High Aromatic	CTUD.		
SEAL TYPE			×	-65 to 275			Buna N							Pet. Lub. & Hyd. Oils		[
			×	-65 to 160			Neoprene							Refrigerants, CO ₂		TYPICAL DESIGNS	
AL YSIS	E.J. Parker	P-010	×	-65 to 300			Natural Rubber	100001					Satisfactory -	Water, Steam Refrigerants, Alcohols, Sili-CO2	cones/castor On		
SEAL ANALYSIS	REFERENCE	NO.	TYPE OF INFORMATION TESTS RECOMMENDATION	TEMPERATURE OF MAX. RANGE	MOTION-TYPE	VELOCITY FPM	MATERIAL SEAL	MATE	PRESSURE PSIA MAX.	RANGE	LOADING SPRING CONIPRESSION	SURFACE FIN. (μ in CLA)	LEAK RATE (Atm. cc/sec)	FLUID	TEST DURATION LIFE REMARKS	DESIGN	

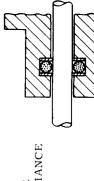
15HES
16
64 TO 125

88	5.1	Nov. 1959 I-012	×	1			Acryle-Nitrile Rubber and	Chloroprene	4000		<i>b</i> 's or		No Leakage		A G			^	_
TABLE		Feb. 1953	×	-65 to 75F			AF934	O-Ring					No Leakage	100% Isooctane 70% Isooctane-	30% PhMe	of Days	O-Ring Inter-	connects-ruel Resistance	lests
	M.W. Varrell	May 1957 AD140116	×	1000			#73902-5 Ring	Pkg. Plug-Sonar Stuffing Tube					Satisfactory	Water		-		<u> </u>	_
O-Ring (Static)			×	-20 to 250			"V"yram						•	. Phosphate	Fluid, Organic Phosphates, Pet. Phosphates, Pet.				_
SEAL TYPE			×	-20 to 400			Poly-FBA								Fluid, Organic Phosphates, Pet		4.5 (Where ¢< 0.15)		W = Compressive Load (kg) d - Material Diam, (cm) $\{\epsilon : \delta / d$
SEAI			×	-20 to 500+			Viton A							Broad Range of Fluids			х є ^{1.3} х н		W = Compres d - Material E · S /d
			×	-20 to 350				Elastomer						Fuming Nitric Acid			$8 \times 10^{-7} \times d^{1.2}$	(Source I-012)	
SEAL ANALYSIS	E.J. Parker	P-010	×	-320 to 250			KEL-F and	Fluoroethane 					Satisfactory	Various Chemicals			$W/L_0 = 4.8 \times 10^{-7}$	<i>x</i> .	
SEAL A	REFERENCE	<u>NO.</u>	TYPE OF INFORMATION TESTS RECOMMENDATION	TEMPERATURE OF MAN. RANGE	MOTION-TYPE	VELOCITY FPM	MATERIAL SEAL	MATE	PRESSURE PSIA MAX.	RANGE	LOADING SPRING COMPRESSION	SURFACE FIN. (µ in.CLA)	LEAK RATE (Atm. cc/sec)	FLUID	TEST DURATION	LIFE	REMARKS		DESIGN

5 - Deformation Magnitude (cm) H - JIS Hardness Lo - Outer RingCircumference (cm)

SEAL A	SEAL ANALYSIS		SEAL	SEAL TYPE	O-Ring (Static)			TABLE	Ċ.
REFERENCE	F. R. Straus								
NO.	Jan. 1955 AD74089					AD11484	AD11484	AD145846	AD145846
TYPE OF INFORMATION TESTS RECOMMENDATION	×	×	×	×	×	×	×	×	×
TEMPERATURE OF MAN. RANGE	-40	- 40	-65 to 400	200	550	-65 to 70	-65 to 70	300	212
MOTION-TYPE VELOCITY FPM	Preconditioned with 40,000 Cycles at 300F	ith 40, 000							
MATERIAL	Neoprene WRT- Spiral Teflon		453-26C	453-9A	453-9A	AF934	AF934	Silicone W96	Silicone W96
MATE	back-up Kings on Both Sides		O-Ring		1203-70				
PRESSURE PSIA MAX.	25	3000	Compound						
RANGE									
LOADING SPRING COMPRESSION									
SURFACE FIN. (µ in.CLA)									
LEAK RATE (Atm. cc/sec)	0	700.	Satisfactory			0	0		
FLUID	ML08200 Hydraulic Oil	lic Oil				Isooctane	70% Isooctane 30% PhMe	70% Isooctane ASTM Oil No. 2 ASTM Oil No. 30% PhMe	ASTM Oil No. 3
TEST DURATION LIFE REMARKS	20 Hrs.	1 Hr.		5 Hrs.	1, 5 Hrs,	37 days	37 days	Satisfactory	Satisfactory
DESIGN					Z				





SEAL TYPE "V"-Ring (Dynamic)		00	A		.Rub.							S	ir	
S	×	-20 to 600			Fab.& Syn. (Fabricat	Metal	8000			32		}	Just Fair	
	×	500 -65 to 400			Synth. Rubber Fab.& Syn.Rub. (Homogeneous) (Fabricated)	Metal	2000			16		Oil, Water, Air Steam, Solvents	Acid, Alkalies	
SEAL ANALYSIS J.N. Smith Jan. 1961 S-070	×	-65 to 180	Reciprocating		Leather	Metal	125,000			63		Oil Water	Air	
SEAL AN REFERENCE NO.	TYPE OF INFORMATION TESTS RECOMMENDATION	TEMPERATURE ^O F MAX. RANGE	MOTION-TYPE	VELOCITY FPM	MATERIAL SEAL	MATE	PRESSURE PSIA MAX.	RANGE	LOADING SPRING COMPRESSION	SURFACE FIN. (μ in CLA)	LEAK RATE(Atm. cc/sec)	FLUID	TEST DURATION LIFE REMARKS	

21

												General Reference - Encapsulants:	Investigating Encapsulants For Ground Electronic Equipment.	June 1956 - Battelle Memorial Institute
E.T. Leidigm et al June 1957 AD140572	×	Cycling -65 to 160F		Polyurathane Foam						No exudation		4 Weeks	Bomb Sealer	
House, P.A. Feb. 1962 AD273284	×				Triethylene Tetramine					Seals punctures caused	oy se camer buners		Punctured Wall Sealer	Hollow wall containing two fluids separated by membrane. Puncture causes fluids to mix and solidify.
Islinger, J.S. June 1956 AD106754	×	-85 to 200		Silicone Rubber		27		Inflated		Seal out	Air	1000 Open & Close Cycles At 75F & 25	At -65F Aircraft	Callopy Scal
REFERENCE NO.	TYPE OF INFORMATION TESTS RECOMMENDATION	TEMPERATURE OF MAX. RANGE MOTION-TYPE	VELOCITY FPM	MATERIAL	MATE	PRESSURE PSIA MAX.	RANGE	LOADING SPRING COMPRESSION	SURFACE FIN. (in. CLA)	LEAK RATE (Atms. cc/sec)	FLUID	TEST DURATION LIFE REMARKS	1	DESIGN



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Rocket Engines"	(Vol. 1.) Vol. 2	Fol 3A, and VOL 3B
Originating Organization: Advanced	Machael and a	OI SA, and VOL 3B
Originating Organization: Advanced Identification No. NASA NAS7-102		
	Date of Do	period 2/26/62 - 2/25/63
Contract, subcontract, or grant no.		/
Security classification of document	UNCL of ti	ttle <u>UNCL</u> of abstract <u>UNCL</u> .
(2 copies each volume to AFSS-A)	Submitted	by Henry Burlage, Jr. Ode RPL (name and ode of office)
	Date	July 11, 1963
OSTI Test Form: January 1963 F.	. W. Kemmett	
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